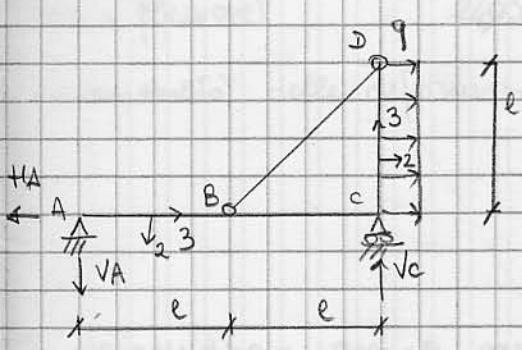


$$l = 3 \text{ m}, q = 1 \text{ t/m},$$
$$\sigma_{\text{AMM}} = 2400 \text{ kg/cm}^2, E = 2.1 \cdot 10^6 \text{ kg/cm}^2$$
$$\Delta T = 20^\circ\text{C}, \alpha = 10^{-5} \text{ }^\circ\text{C}^{-1}$$

La travatura iperstatica di figura è realizzata con profilati IPE.

1. Utilizzando il metodo delle forze risolvere la travatura in presenza del solo carico q e disegnare i diagrammi delle caratteristiche di sollecitazione (N , T , M).
2. Progettare la travatura.
3. Calcolare la rotazione del nodo B .
4. Risolvere nuovamente la travatura considerando anche il carico termico nella biella BD . Disegnare i nuovi diagrammi delle caratteristiche di sollecitazione (N , T , M) comprensivi sia di q che del carico termico.

B.1)



$(\rightarrow) H_A = qe$

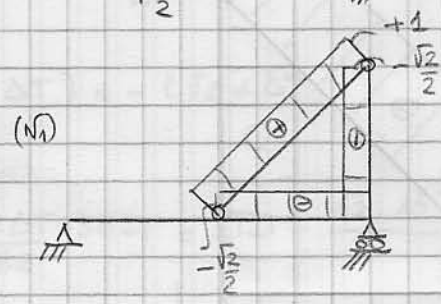
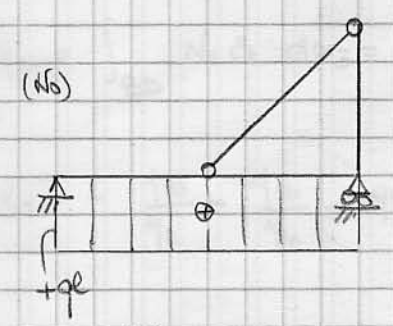
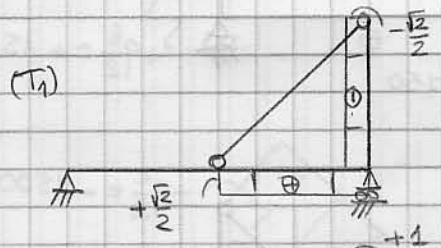
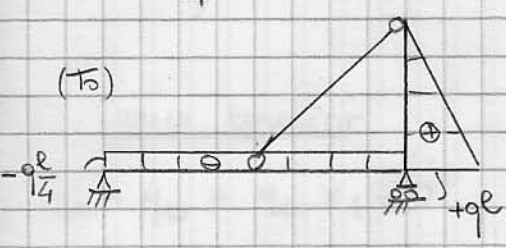
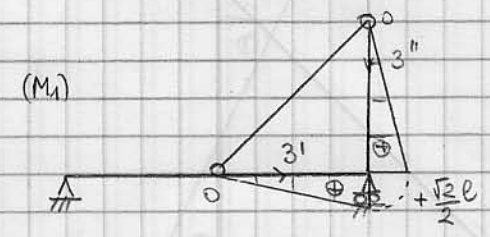
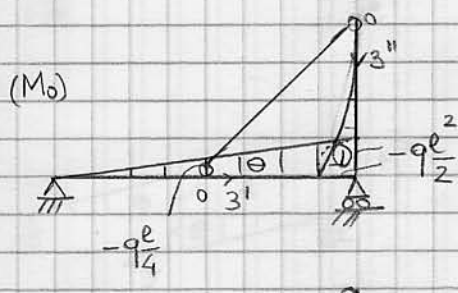
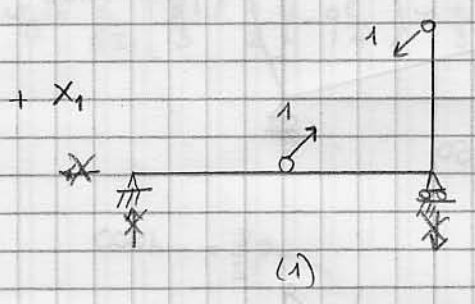
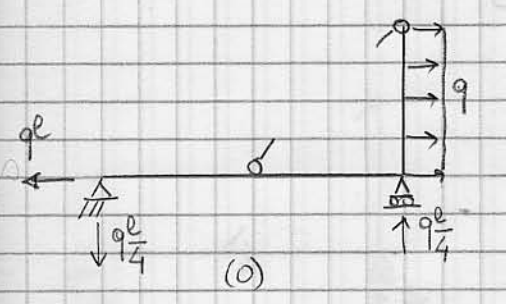
$(\uparrow) V_A = V_c$

$(A\uparrow) V_c 2e = q \frac{e^2}{2} \rightarrow V_c = \frac{qe}{4}$

$\begin{cases} H_A = qe \\ V_A = V_c = \frac{qe}{4} \end{cases}$

La trave e' isostatica per vincoli esterni.

Internamente e' una volta iperstatica: $X_1 = N_{BD}$



Eq. me di Muller-Breslau: $0 = M_{10} + M_{11} X_1$

$EI_1 M_{10} = \int_0^e (-\frac{qe^2}{4} - \frac{qe}{4} x_3') (\frac{\sqrt{2}}{2} x_3') dx_3' + \int_0^e (-\frac{qx_3''}{2}) (\frac{\sqrt{2}}{2} x_3'') dx_3''$

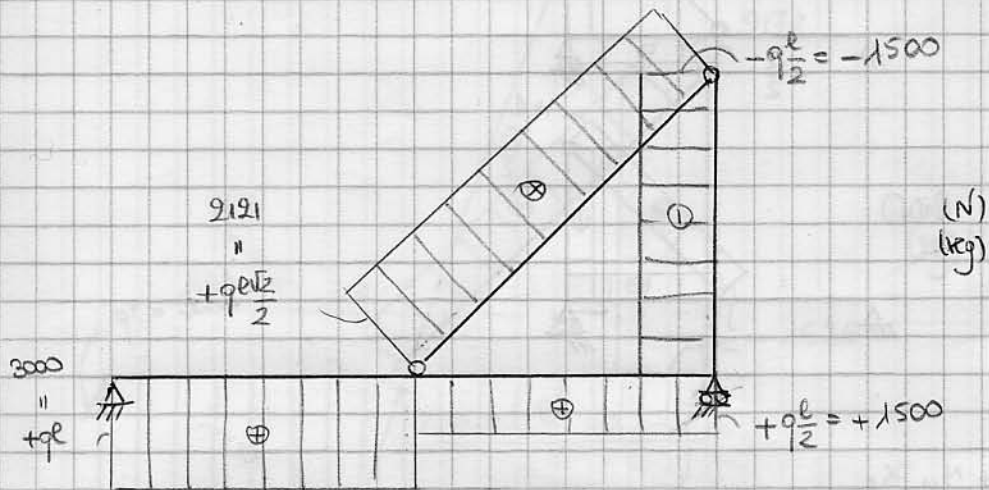
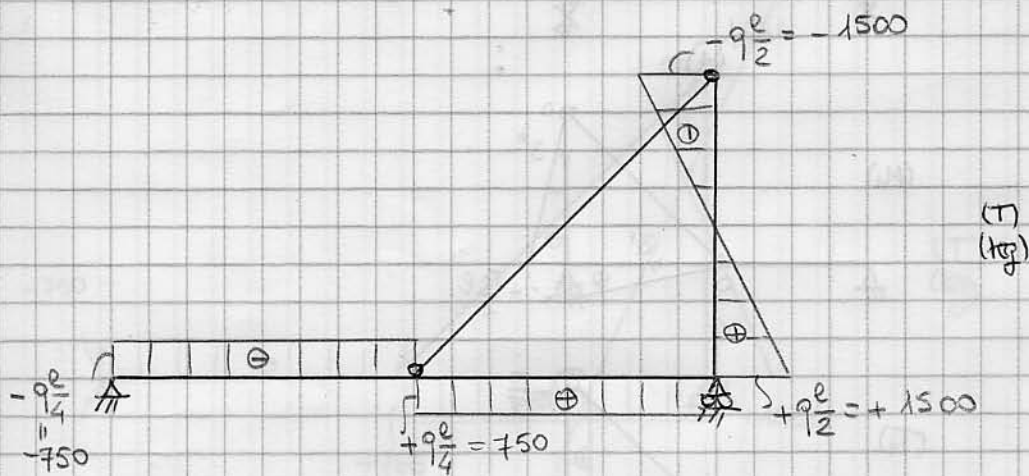
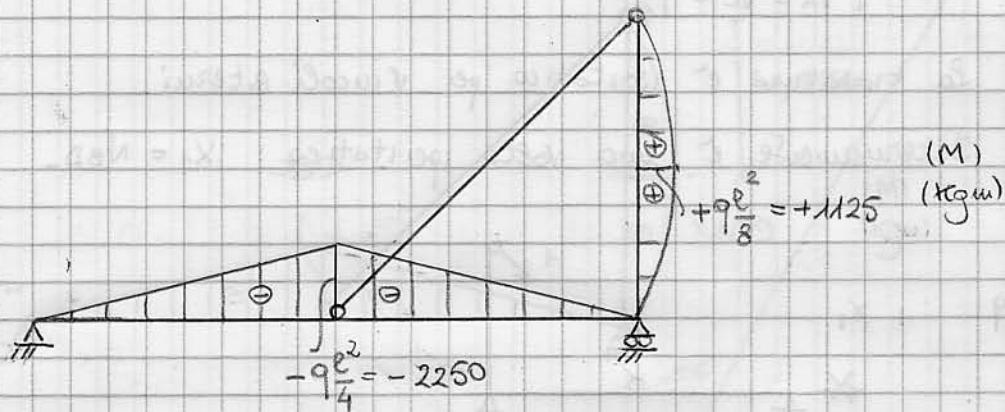
$= -\frac{qe^2 \sqrt{2}}{4} \int_0^e (ex_3' + x_3'^2) dx_3' - \frac{q\sqrt{2}}{8} \int_0^e x_3''^2 dx_3'' = -\frac{qe^2 \sqrt{2}}{4} (\frac{e^2}{2} + \frac{e^3}{3}) - \frac{q\sqrt{2} e^3}{24} = -\frac{qe^2 \sqrt{2}}{8} (\frac{1}{2} + \frac{1}{3} + \frac{1}{2})$

$= -\frac{1}{3} \frac{qe^2 \sqrt{2}}{8} = -\frac{qe^2 \sqrt{2}}{6}$

$EI_1 M_{11} = \frac{q \cdot 1}{3} e (\frac{\sqrt{2} e^2}{2}) = \frac{e^3}{3}$

$$X_1 = -\frac{M_{10}}{M_{11}} = qe \frac{\sqrt{2}}{6} \frac{3}{2^2} = qe \frac{\sqrt{2}}{2} = 2121 \text{ kg}$$

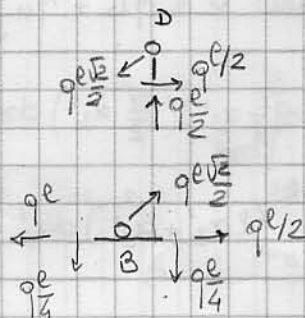
Diagrammi quotati:



$$M_C = -\frac{qe^2}{2} + \frac{\sqrt{2}}{2} e X_1 = -\frac{qe^2}{2} + \frac{\sqrt{2}}{2} e qe \frac{\sqrt{2}}{2} = 0$$

$$T_C = qe - \frac{\sqrt{2}}{2} X_1 = qe - \frac{qe}{2} = \frac{qe}{2}$$

$$T_D = -\frac{\sqrt{2}}{2} X_1 = -\frac{qe}{2}$$



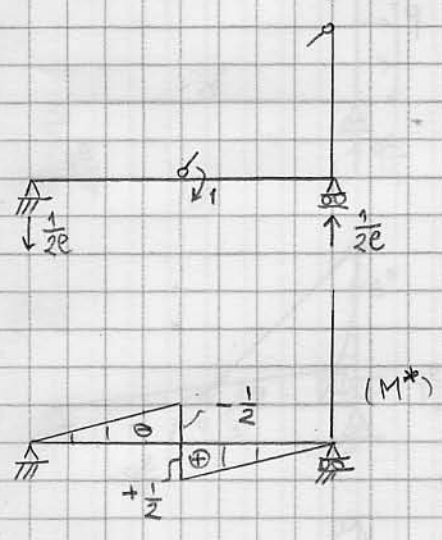
A.2) Progetto : $W_1 \geq \frac{2250 \cdot 100}{2400} = 93,75 \text{ cm}^3$ IPE 160 $\begin{cases} A = 90,1 \text{ cm}^2 \\ I_1 = 869 \text{ cm}^4 \end{cases}$
 (solo a flessione)

Trasversale delle deformazioni assiali: $\frac{\Delta L}{L} = (\sqrt{2} \cdot 1 + 2 \cdot \frac{1}{2}) \frac{e}{EA} \cdot \frac{3EI}{l^3}$
 $= (1 + \sqrt{2}) \frac{3I_1}{AB^2} = 0,35 \%$

Le deformazioni assiali sono trascurabili.

A.3) Rotazione in B

$$1 \cdot \varphi_B = \frac{1}{EI_1} \left[\frac{1}{3} l \frac{1}{2} (-9 \frac{ql^2}{4}) + \frac{1}{3} l \left(-\frac{1}{2}\right) \left(-9 \frac{ql^2}{4}\right) \right] = 0$$



A.4) Carico termico

$$M_{11} + M_{10} + M_{11} X_1 = 0$$

$$M_{11} = \int_{BD} N_1 \epsilon_t dx_3 = l\sqrt{2} \cdot (-\alpha \Delta T) = -l\sqrt{2} \alpha \Delta T$$

$$X_1 = -\frac{M_{10}}{M_{11}} - \frac{M_{11}}{M_{11}} = \frac{9ql^2}{2} + \frac{l\sqrt{2} \alpha \Delta T}{l^2} 3EI_1 = 2121 + \frac{\sqrt{2} \cdot 10^{-5} \cdot 20 \cdot 3 \cdot 2,1 \cdot 10 \cdot 869}{9 \cdot 10^9}$$

$$= 2121 + 17 = 2138 \text{ kg}$$

Il diagramma si discosta poco da quello rappresentato al pto (1)