

ρ

$$F(\rho, p, T) = 0 \quad \text{gen}$$

$$G(\rho, p) = 0 \quad \text{bar}$$

$$\rho = \text{const}$$



$$\rho \bar{f} - \nabla p = 0$$

$$\bar{f} = \nabla \phi \quad \bar{f} \text{ cons.}$$

$$\textcircled{1} \quad \bar{f} \rightarrow 0 \quad (\text{GAS})$$
$$p = \text{const}$$

$$\textcircled{2} \quad \rho = \rho(p)$$

BAROTR.

$$\phi = -gz + \text{const}$$

$$h = \underbrace{z} + \underbrace{\int \frac{dp}{\rho g}} = \text{const}$$

③

FL. INCOMPRESS.

$$\rho = \text{cost}$$

$$\gamma = \rho g$$

$$h = z + \frac{p}{\gamma} = \text{cost}$$

IL CARICO PIEZOMETRICO IN
UN FLUIDO INCOMPRESS. NEL
CAMPO GRAV.

E' COSTANTE

$$z + \int \frac{dp}{\rho g} = \text{cost}$$

$$0 \quad p_0 \quad p \quad p(z_0) = p_0$$

$$\int_{p_0}^p \frac{dp}{\rho g} = -(z - z_0)$$

$\uparrow$
 $p(p)$

$$\frac{p}{\rho} = RT$$

EQ. NE di STATO

$$T = \text{cost.}$$

$$\rho(p_0) = \rho_0$$

$$\frac{p_0}{\rho_0} = RT_0$$

$$\boxed{\rho = \rho_0 \, p/p_0}$$

$$\rho = \rho(p)$$

$$\frac{dp}{p}$$

$$\int_{p_0}^p \frac{dp}{\rho_0 \, p/p_0 \, g}$$

$$\ln p - \ln p_0 = \ln(p/p_0)$$

$$\left(\frac{p_0}{\rho_0 g} \right) \ln(p/p_0) = -(z - z_0)$$

$$* \frac{p}{p_0} = \exp \left\{ - \frac{\cancel{p_0 g}}{p_0} (z - z_0) \right\}$$

$$e^{-x} \simeq 1 - x + \frac{x^2}{2} + o(x^2)$$

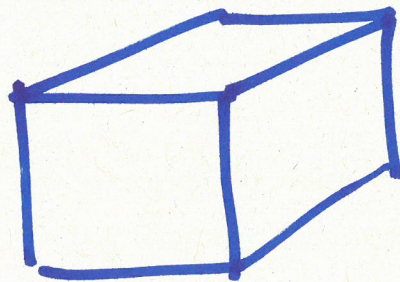
$$z \rightarrow z_0$$

$$\frac{p}{p_0} = 1 - \frac{\gamma_0}{p_0} (z - z_0) + \left[\left(\frac{\gamma_0}{p_0} \right) (z - z_0) \right]^2 \cdot \frac{1}{2} + \dots$$

$$\frac{p}{p_0} = 1 \quad \text{se } |z - z_0| \lesssim 70 \text{ m}$$

$$\text{err} < 1\%$$

$$p = p_0 = \text{const}$$



$$\vec{f} = 0$$

$$\text{se } |z - z_0| \in [70 \text{ m}; 1000 \text{ m}]$$

$$\frac{p}{p_0} = 1 - \frac{\gamma_0}{p_0} (z - z_0)$$

$$p = p_0 - \gamma_0 (z - z_0)$$

$$\gamma = \rho g$$

$$\gamma = \gamma_0 = \text{cost}$$

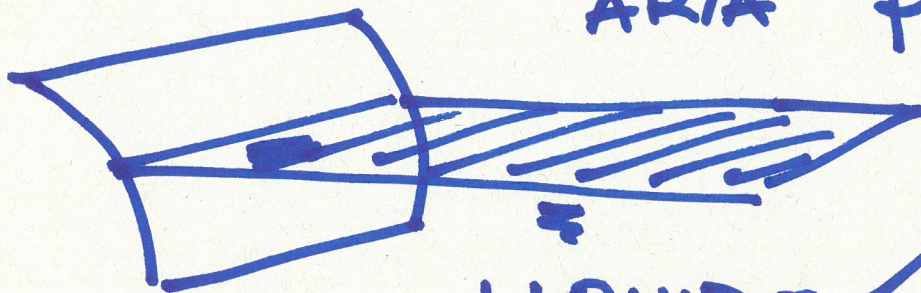
$$p = p_0 = \text{cost}$$

$$|z - z_0| > 1000$$

$\rightsquigarrow$ gas stratificato

$$\left(\right) \frac{p}{\rho^k} = \text{cost}$$

$$\text{ARIA } p = p_0 = \text{cost}$$



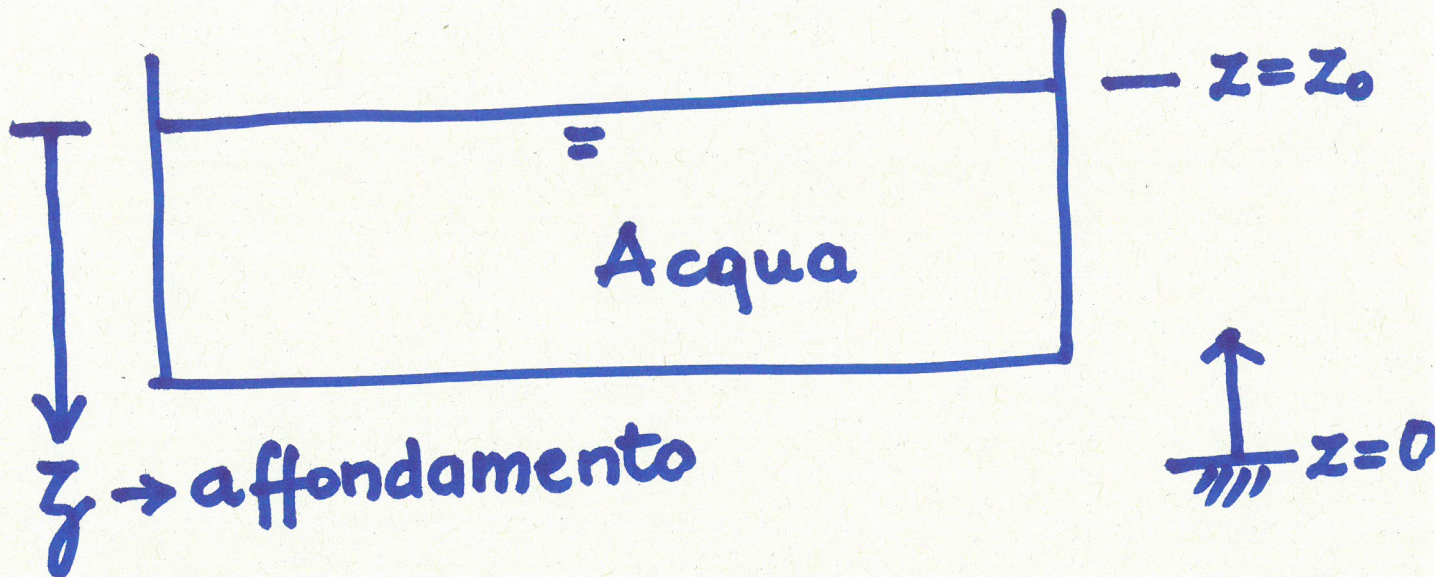
LIQUIDO INCOMPR.

$\downarrow$ distr. della press.
idrostatica

$$p - p_0 = -\gamma (z - z_0)$$

$$\rho = \text{cost}$$

Aria



$$\phi \quad p - p_0 = -\gamma (z - z_0)$$

$$p_{\text{atm}}^* = 1.01325 \text{ bar} \quad \frac{1 \text{ N}}{\text{m}^2}$$

$$= 1.013 \cdot 10^5 \text{ Pa}$$

$$p_{\text{rel}} = p^* - p_{\text{atm}}^*$$

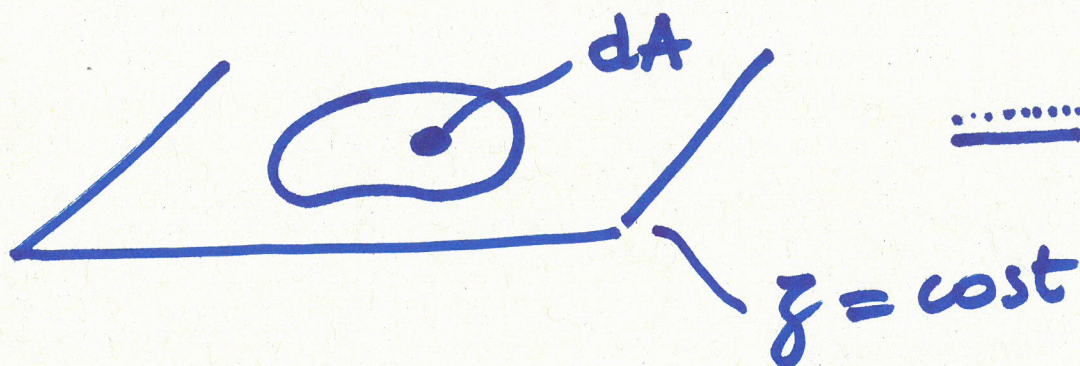
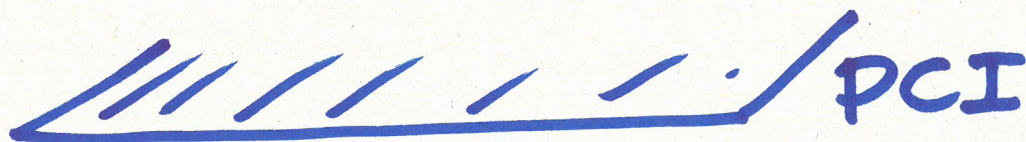
→ p

$$p = \gamma z$$

distr. idr.
di pressione
fl. inc. in quiete
nel campo gr.

$$\frac{p^*_{atm}}{\gamma} = 10,33m$$

SPINTA IDROST. SU SUP.
PIANE

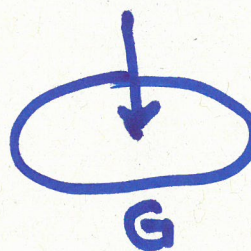


$$\delta \vec{F} = p \delta A \vec{n}$$

A small diagram showing a circle with a downward-pointing arrow, representing a differential force vector.

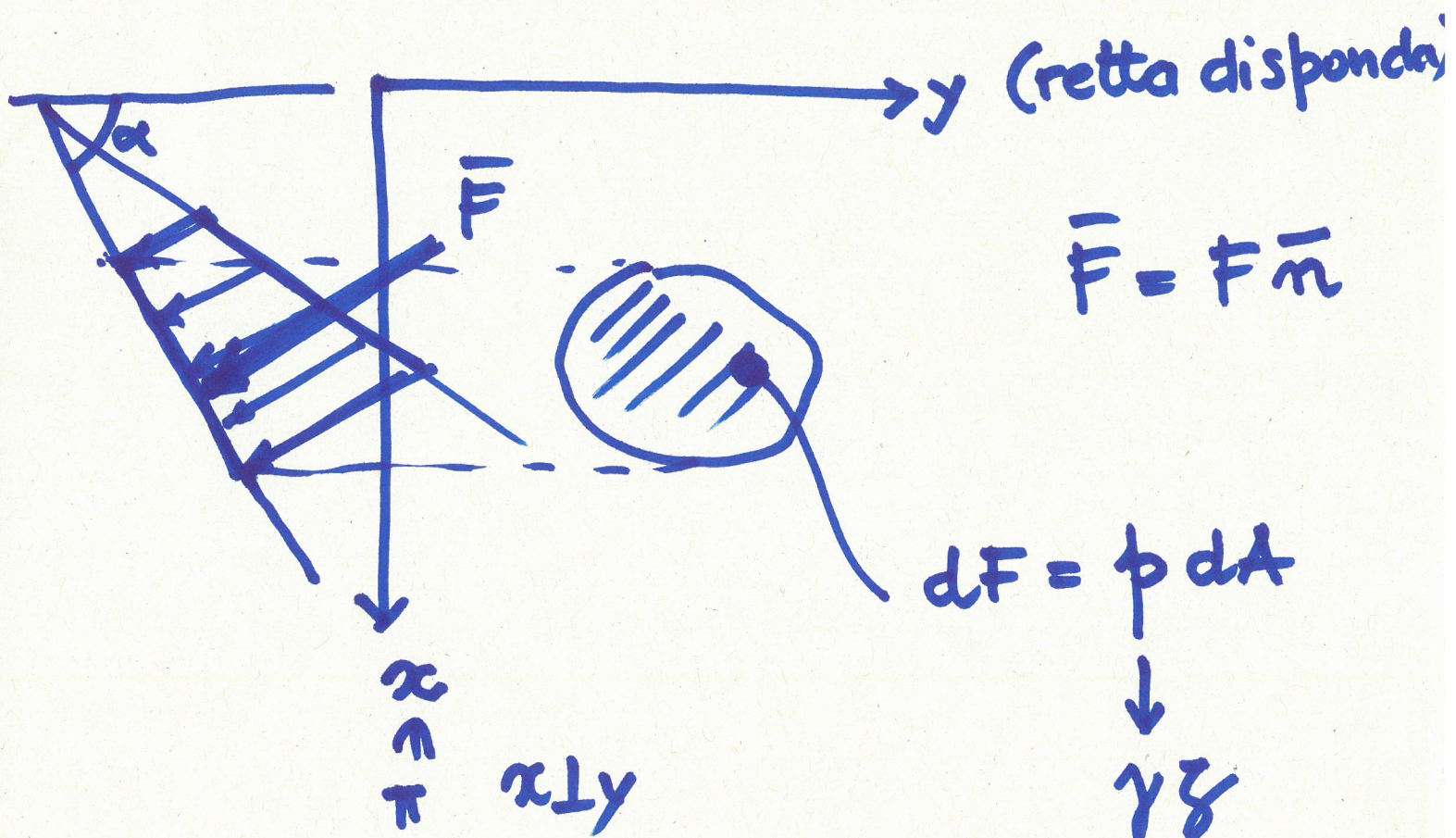
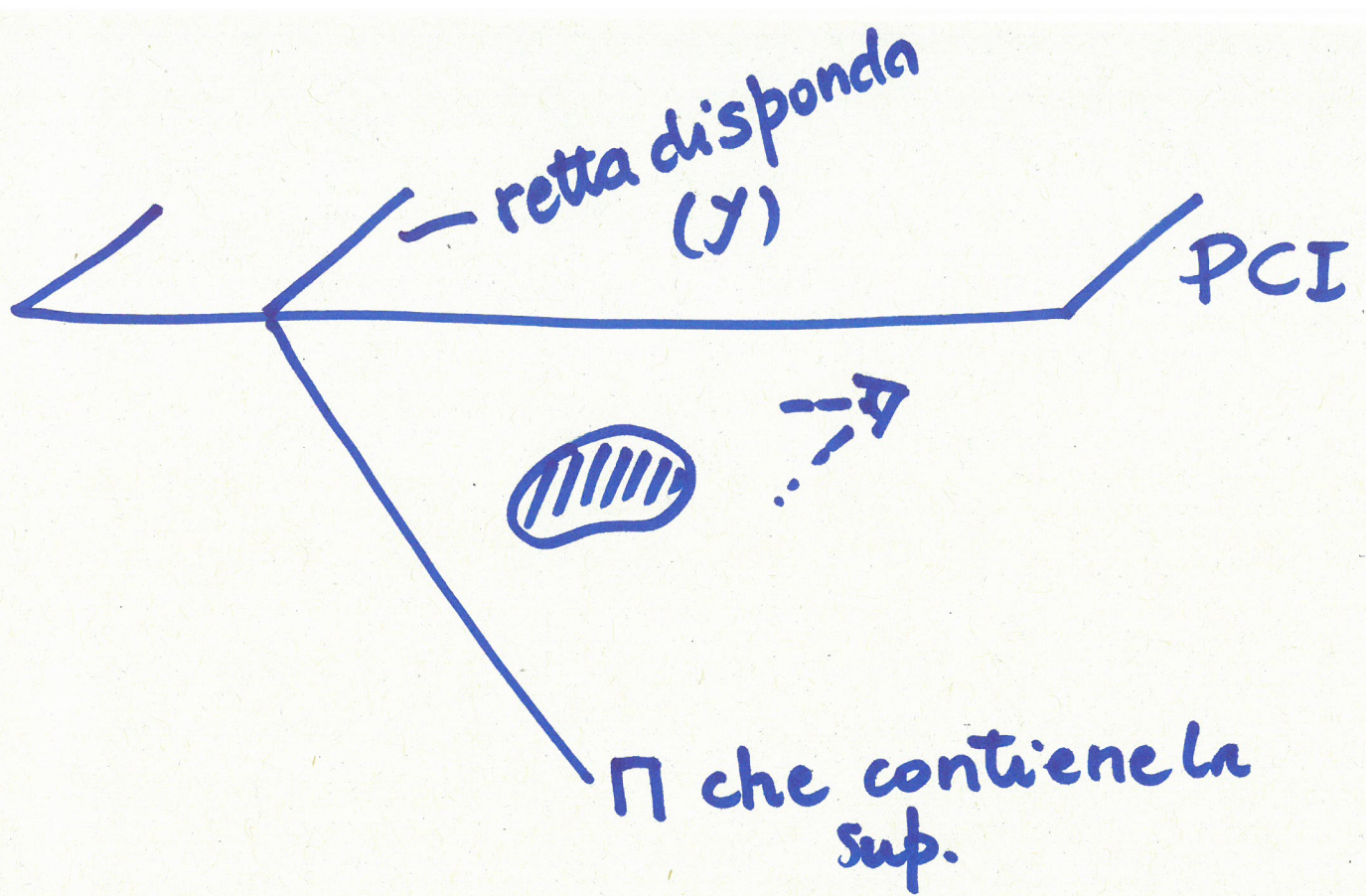


$$\vec{F} = F \vec{n}$$

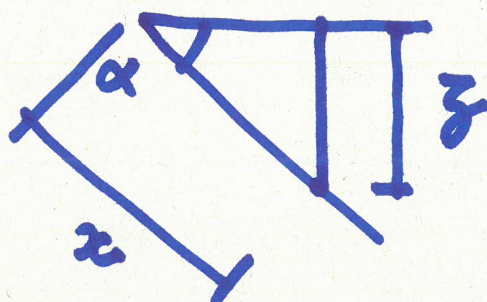


retta di appl. per G

$$F = \int_A dF = \int_A \underbrace{p}_{\gamma z} dA = \underbrace{\gamma z_G}_{p_G} A$$



$$z = x \sin \alpha$$



$$dF = \gamma \delta dA = \gamma x \sin \alpha dA$$

$$F = \int_A dF = \int_A \gamma x \sin \alpha dA$$

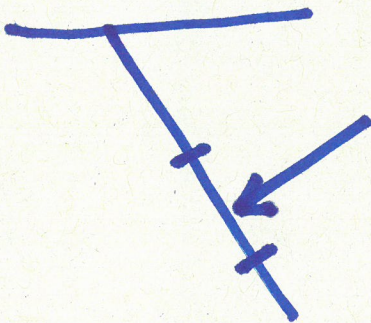
$$= \cancel{\gamma} \cancel{\sin \alpha} \int_A x dA$$

$x_G A$

$$F = \gamma x_G \sin \alpha A = \gamma \cancel{x_G} A$$

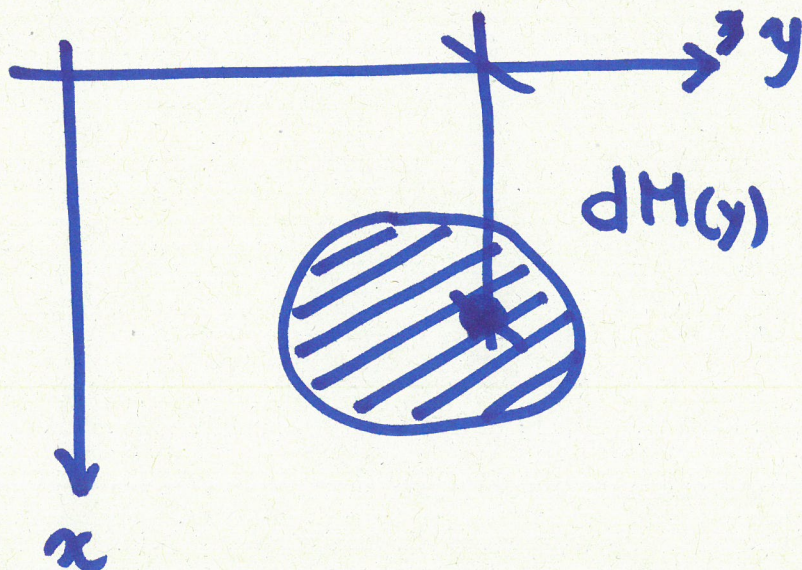
$$= \cancel{\rho} G A \quad \swarrow \text{mod.}$$

$$\boxed{S_y = x_G A}$$

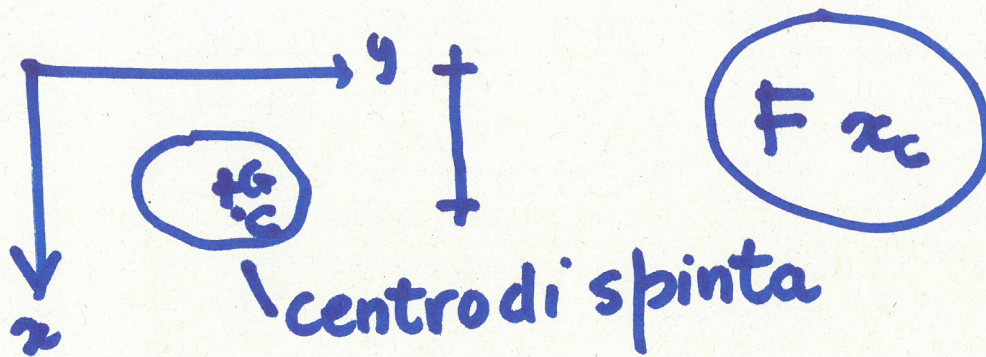


$$\vec{F} = F \vec{n}$$

verso contro
direz $\perp$



$$dM_y = dF \cdot x = \gamma x^2 \sin \alpha dA$$



$$F x_c = \int_A \gamma x^2 \sin \alpha dA$$

$$\gamma S_y x_c \sin \alpha = \gamma \sin \alpha \int_A x^2 dA$$

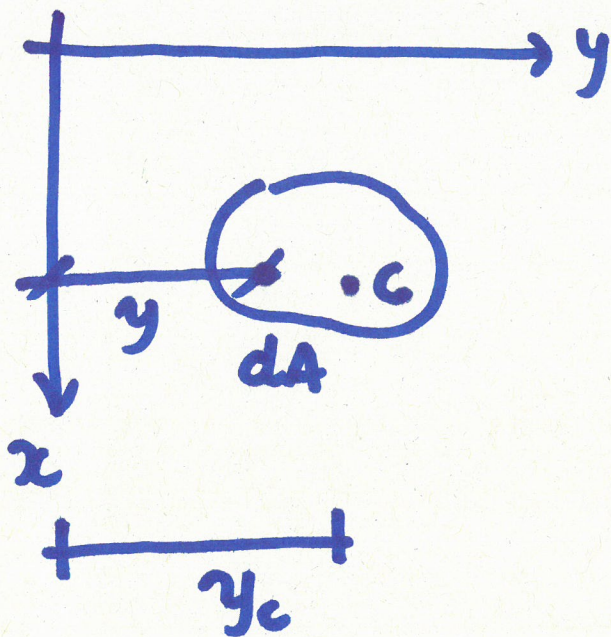
J_{yy}

$(x_c A) \sin \alpha$

$$x_c = \frac{J_{yy}}{S_y}$$

La retta d'azione passa per un punto la cui coordinata x_c

è il rapporto tra mom. di inerzia e momento statico della fig. risp. all'asse y



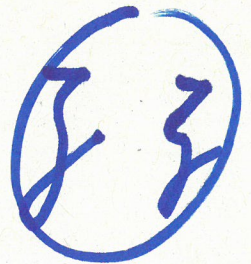
$$F y_c$$

$$\underbrace{\gamma z dA y}_{dF}$$

$$F y_c = \int_A \underbrace{\gamma z}_{\gamma \sin \alpha} dA y$$

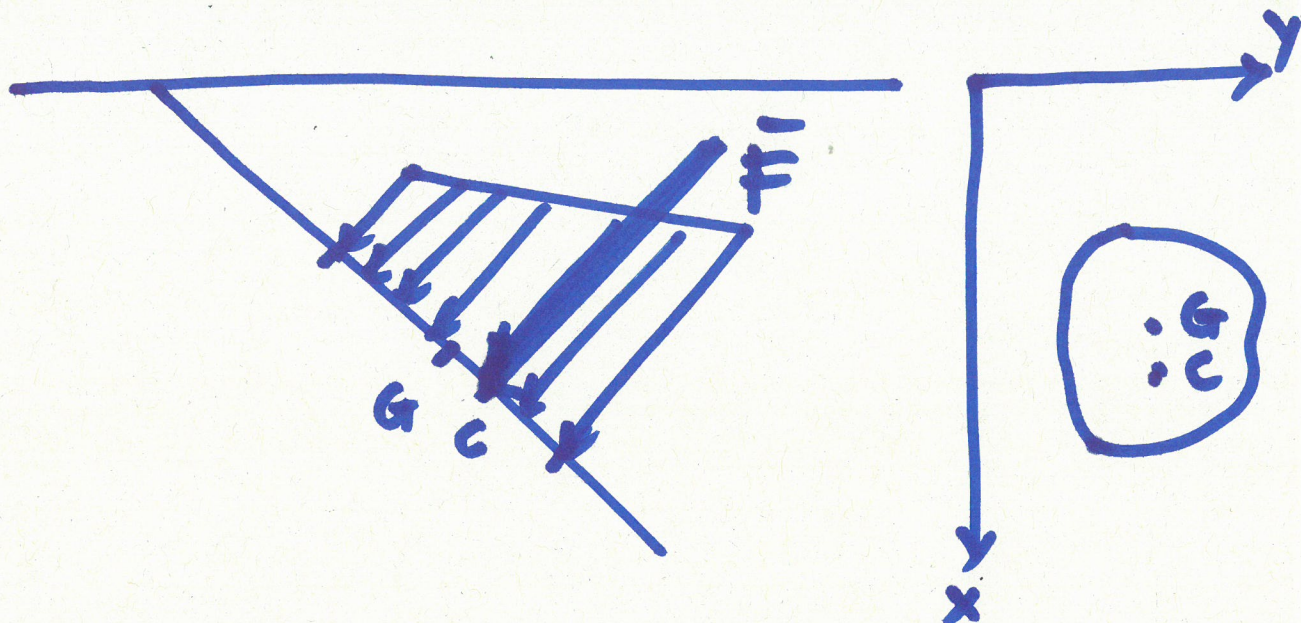
$$\gamma \underbrace{x_c A \sin \alpha}_{S_y} y_c = \gamma \sin \alpha \int_A x y dA$$

$$S_y y_c = \int_A x y dA$$



$$y_c = \frac{J_{xy}}{S_y}$$

La retta d'azione passa per un punto la cui coordinata y_c è il rapporto tra il momento di inerzia dev. e il mom. st. y



$$\bar{F} = F \bar{n}$$

direz $\perp$ alla sup
verso contro la sup

$$F = \gamma \bar{z}_G A$$

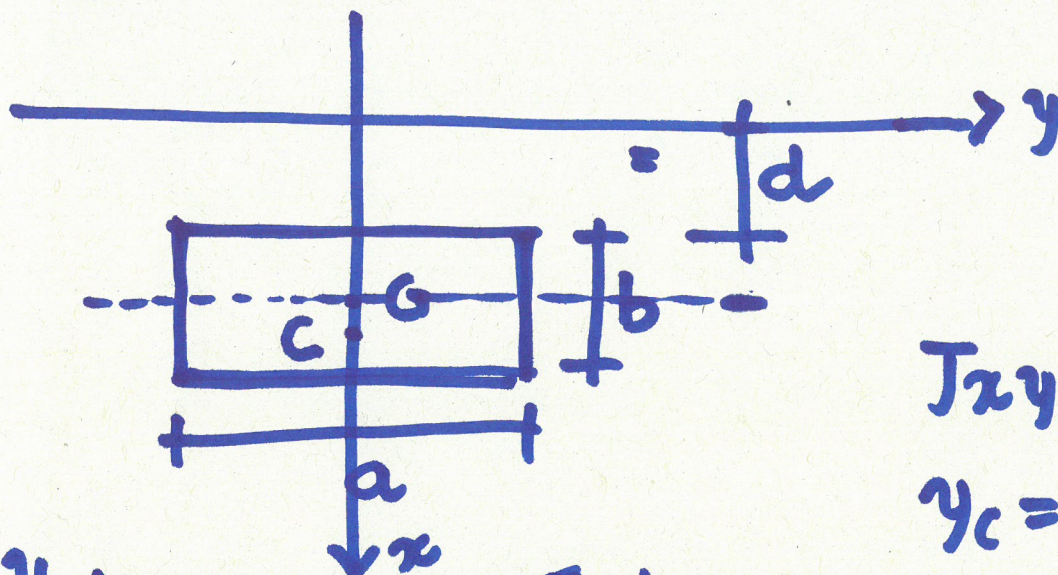
coord. punto di appl.

$$\begin{cases} x_c = \frac{J_{yy}}{S_y} \\ y_c = \frac{J_{xy}}{S_y} \end{cases}$$

$$S_y = x_G A$$

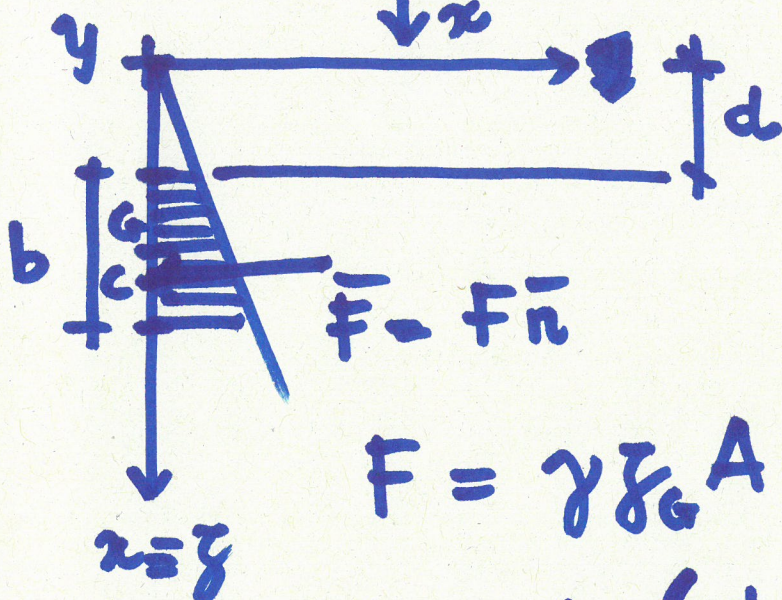
$$J_{yy} = \int_A x^2 dA \quad J_{xy} = \int_A xy dA$$

$$\text{se } \alpha = \pi/2 \quad \boxed{x \equiv \bar{z}}$$



$$J_{xy} = 0$$

$$y_G = y_C = 0$$



$$F = \gamma \bar{x}_G A$$

$$= \gamma \left(d + \frac{b}{2} \right) (ab)$$

$$x_c = \bar{x}_c = \frac{J_{yy}}{S_y}$$

$$J_{yy} = J_{yy_G} + A x_G^2$$

Teorema
di
↑
Huygens

* Il mom. di inerzia rispetto all'asse y è uguale al mom. di in. rispetto ad un asse // all'asse y e baricentrico + area · (dist. baric. asse)

$$J_{yyG} = \frac{1}{12} a b^3$$

$$x_c = \zeta_c = \frac{(ab)(d+b/2)^2}{(d+b/2)(ab)} +$$

$$+ \frac{1}{12} \frac{a b^3}{(d+b/2)(ab)}$$

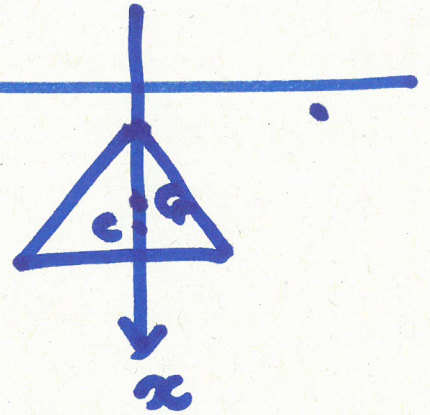
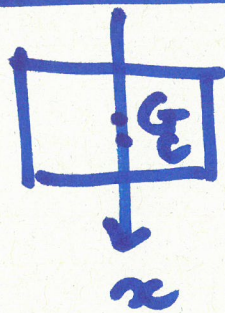
$$\zeta_c = \underbrace{d+b/2}_{x_G} + \frac{1}{12} \frac{b^2}{(d+b/2)}$$

$$x_c = x_G + \frac{J_{yyG}}{A x_G}$$

IL CENTRO DI SPINTA
E' PIÙ AFFONDATO
DEL BARICENTRO!

Se possibile, usare asse x già baricentrico.

→ CASI SIMM

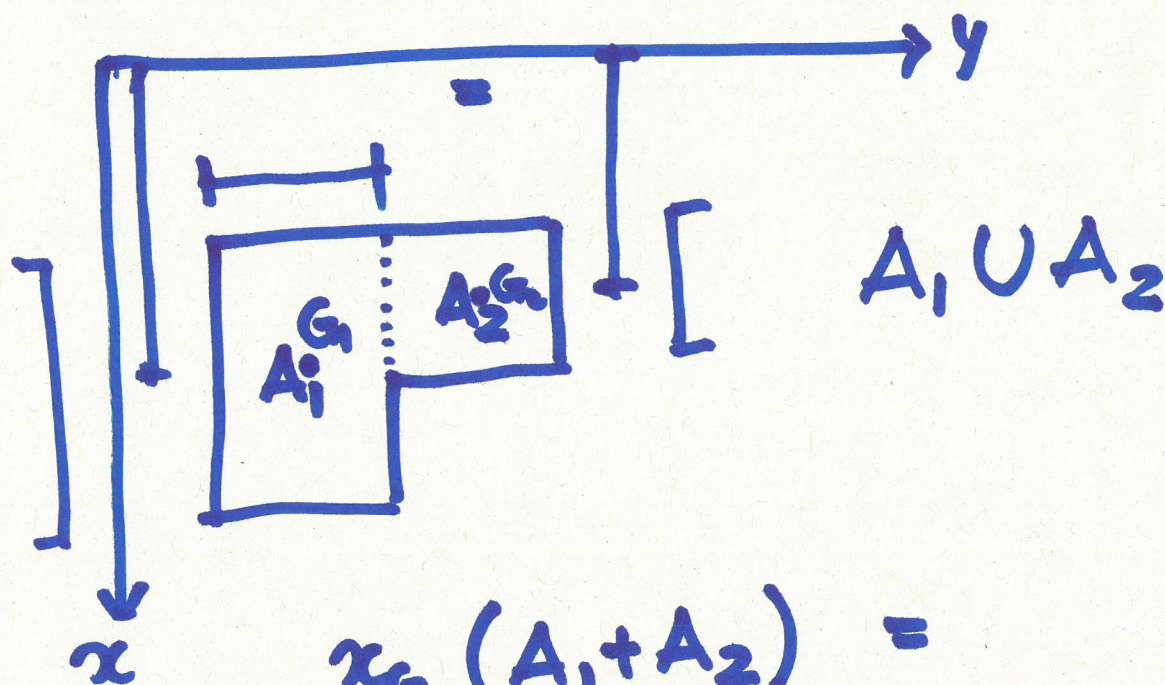
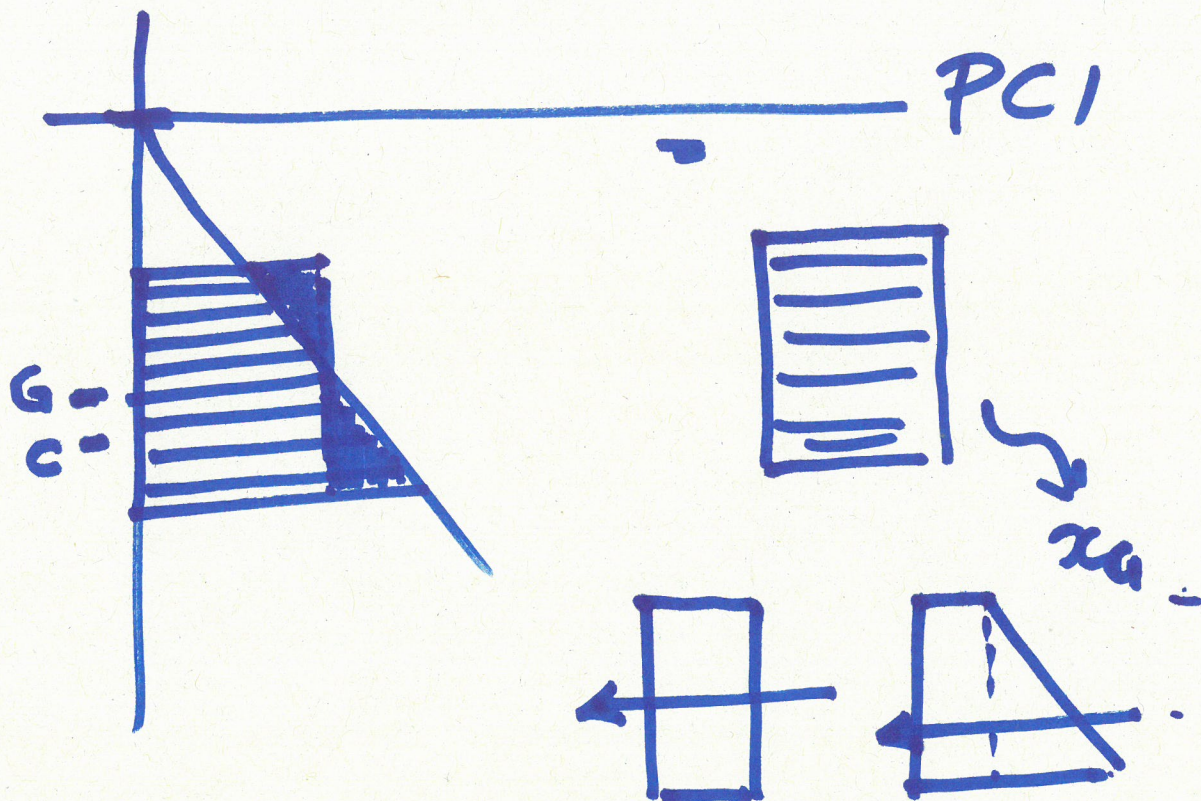


CALCOLO DEL MODULO DELLA FORZA: COMPARE x_G (J_G)

LA FORZA NON E' APPLICATA IN G, MA IN C (più affond. di G)

SE x è di simmetria,

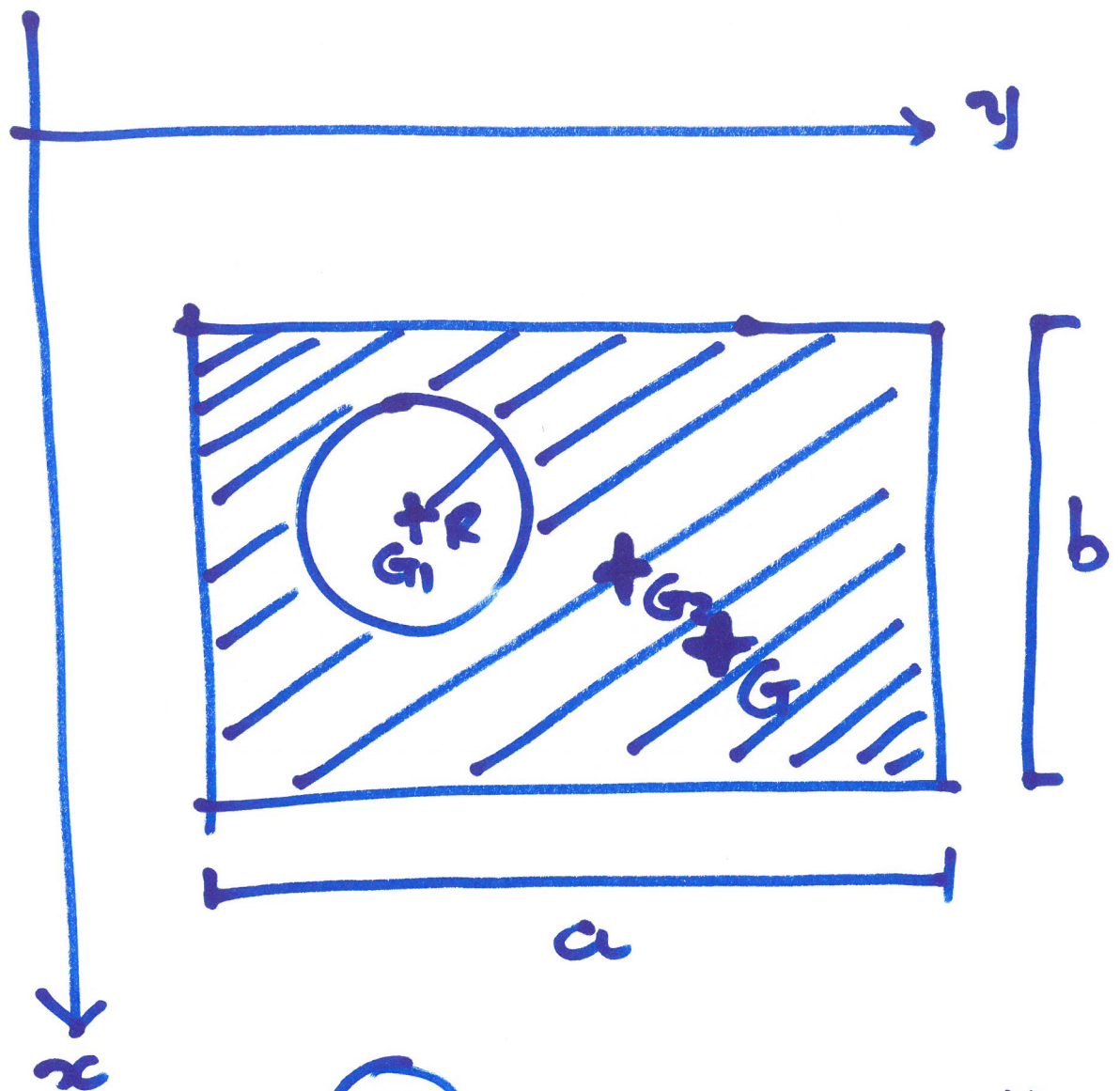
$$y_G = y_C = 0$$



$$x_G (A_1 + A_2) =$$

$$= x_{G1} A_1 + x_{G2} A_2$$

$$x_G = \frac{A_1}{A_1 + A_2} x_{G1} + \frac{A_2}{A_1 + A_2} x_{G2}$$



$G_2 = G_{\text{rett pieno}}$

$$x_{G_2} A_{\text{rett}} - x_{G_1} A_c$$

$$= \textcircled{x_G} (A_{\text{rett}} - A_c)$$

↓ ?